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# A Novel (3T-2R) Parallel Mechanism with Large Operational Workspace and Rotational Capability\*

Samah Shayya<sup>(a,b)</sup>, Sébastien Krut<sup>(b)</sup>, Olivier Company<sup>(b)</sup>, Cédric Baradat<sup>(a)</sup> and François Pierrot<sup>(b)</sup>

**Abstract**—The paper presents a novel 5 dofs (3T-2R) parallel mechanism. The mechanism is characterized by large singularity-free workspace and particularly large rotational capability which makes it suitable for 5 face-machining and similar applications. Having all its prismatic actuators along x direction, the x-motion is independent from other dofs-only limited by prismatic actuators' strokes- constituting another major advantageous feature. Besides, an analytical direct geometric model can be easily established which is a rare feature in parallel robots. The paper introduces the novel mechanism with its inverse and direct geometric models as well as its kinematic models (forward and inverse Jacobians). Also, it discusses its singularity analysis and presents sample workspace plots evaluated based on isotropic performance regarding velocity and static force.

## I. INTRODUCTION

To cope up with actual industrial needs, a research trend towards the synthesis of lower mobility parallel mechanisms has been noticed in the last decades. In fact, having a 5 dofs (3T-2R) mechanism is sufficient for the majority of industrial applications, such as machining, drilling, laser and water-jet cutting, etc. -even fewer number of dofs in others might be sufficient. These 5 dofs in general can be obtained either in one parallel architecture, via hybridization (parallel-series mechanisms), or through implementing right-hand left-hand paradigm.

Hybridization [1-2] is not a recommendable choice as it increases the moving mass (due to having additional actuators on the platform) and thus impacting dynamical performance as well as the global precision and stiffness of the robot.

On the other hand, right-hand left-hand paradigm might be more recommended and favored over hybridization, especially if 4 dofs are embedded in one parallel module (right-hand) and the remaining 1 dof is provided by separate left-hand mechanism: as for instance the suggested use of ARROW mechanism (3T-1R) studied in [3] with a turntable to provide the 2<sup>nd</sup> rotational dof. Other machines already

exist such as the VERNE machine of [4] (parallel module (3T) + (2R) turntable (series)) and the SPKM 165 of [5] ((2T-1R) parallel module + (1T-1R) table (series)). But unfortunately these [4-5] have rather limited workspace. In addition, such mechanisms are usually less flexible and might introduce challenges regarding control and motion planning. This is not to mention calibration related issues. Thus, in some applications the best solution would be to have 5 dofs (3T-2R) parallel mechanism as one structure.

In literature, there are numerous synthesis and studies on 5 dofs (3T-2R) robots such as in [6-10] and many others... However, most of these available mechanisms are either characterized by limited workspace especially regarding tilting capacity, presence of singularities, and/or complexity of design especially from manufacturability point of view. For example, although the regularity of the mechanisms in [6] is interesting simplifying their analyses, it unfortunately leads to reduced stiffness and precision as not all the kinematic chains cooperate in counteracting the load, and the precision along each axis is rather controlled by individual kinematic chain and individual actuator's resolution. Besides, having prismatic joints along different directions reduces workspace and makes them less practical from industrial point of view. The mechanism in [8] provides 5 dofs (3T-2R) using redundant parallel module formed by two folding five-bar linkages set on two rotatable links and connected to the platform via universal joints and a 7<sup>th</sup> actuator can be added in series to provide the 6<sup>th</sup> dof (3<sup>rd</sup> rotational dof) if needed. The direct geometrical model can be analytically derived which is interesting, but still the rotational capabilities and workspace are limited and dependent on position, not to mention that having the platform held by two universal joints is a drawback as the stiffness might be reduced. Regarding [9], the mechanism is undeniably interesting as it is characterized by large tilting capacity. This not to mention that having the actuators along one direction increases the workspace along this direction. Moreover, having an analytical forward geometric model is an additional bonus. But maybe the only drawback in this mechanism is the presence of cable-pulley or gear mechanism which can impact the robot's performance regarding precision and stiffness.

This paper presents a novel 5 dofs (3T-2R) parallel mechanism in the attempt to resolve the major problematics emphasized in the above argument. The mechanism in its current version implements rack-pinion sub-mechanism (which might indicate similar problems to [9]) to facilitate the study only, but this in fact can be replaced by any rigid linear-to-rotational motion transformation mechanism having no singularity in the required rotational range. Also, compared to that suggested in [9], it has fewer actuators and

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hence lower costs (motors' costs form a very large proportion of the overall robot's cost). The mechanism is characterized by large workspace and tilting capacity, improved kinetostatic performance and relatively high expected dynamic capabilities. The paper introduces the mechanism in (II) with its geometric parameters. The inverse geometric model (IGM) and direct geometric model (DGM) are established in sections (III) and (IV) respectively; while in section (V) the kinematic models and singularities are discussed. Section (VI) then presents sample workspace plots evaluated using isotropic velocity and force based index (same as that used in [3]). The paper is concluded in section (VII) with conclusions and perspectives for future work.

## II. THE NEW MECHANISM AND ITS GEOMETRIC ELEMENTS

The graph diagram of the mechanism with a simplified functional CAD are shown in Fig. 1 and Fig. 2, respectively. Fig. 3 shows the dimensions of the platform and the two moving frames. The functioning of the mechanism is straightforward. Chains (III) through (V) cooperate to position the segment  $B_4B_5$  which is always vertical thanks to the constraining parallelograms (II) and (III); these parallelograms do not allow any rotation about any axis perpendicular to the z-axis of the base frame. Then, chains (I) and (II) together control the two rotational dofs  $\theta_z$  about the z-axis of the base frame and  $\theta_y$  about the y-axis of the frame ( $M_f$ ) (tool frame). This is done thanks to the rack-pinion mechanism shown in Fig.2-3, which by fixing the distance between  $B_1 \equiv B_2$  with respect to parallelogram vertical axis ( $B_4B_5$ ) and rotating it controls the 1<sup>st</sup> rotation (i.e.  $\theta_z$ ). As for  $\theta_y$  it is controlled by moving the rack which turns the pinion an angle proportional to the relative displacement between ( $B_4B_5$ ) and  $B_1$ . Thus, the TCP position and the tool orientation are controlled.

It is worth mentioning that we have used a rack-pinion here for simplicity of the analysis, but in general it can be replaced by any rigid linear-to-rotational motion transformation system having no singularities within the required range of rotation for  $\theta_y$  which is between  $-45^\circ$  and  $+45^\circ$ ; this allows avoiding precision and stiffness problems due to backlashes in rack pinion mechanism as previously highlighted. The mechanism tilting capacities are between  $-90^\circ$  and  $+90^\circ$  regarding  $\theta_z$  and between  $-45^\circ$  and  $+45^\circ$  regarding  $\theta_y$  which are enough for 5-face machining applications...Moreover, the x-motion is independent of the other dofs and only restricted by available stroke for the linear actuators.

Regarding technical issues, it is worth mentioning that the limitation of commercial spherical joints can be overcome by replacing each with three concurrent perpendicular revolute joints.

Finally, we introduce the geometric parameters of the mechanisms as follows:

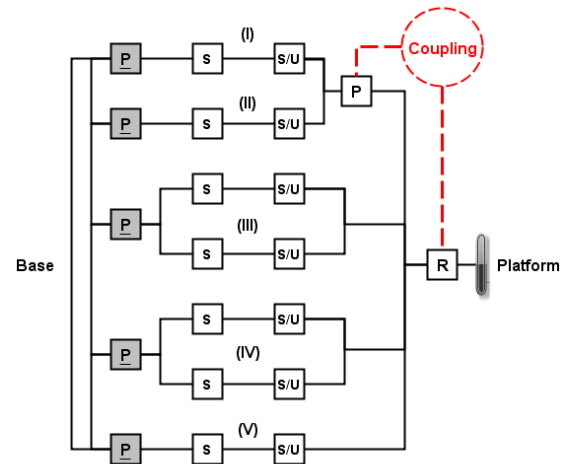


Figure 1. Simplified graph diagram of the 5 dofs (3T-2R) parallel mechanism. P,S,U: stand for prismatic, spherical and universal joints respectively. Gray box signifies that the joint is actuated while underlining means that the joint is equipped with a position sensor.

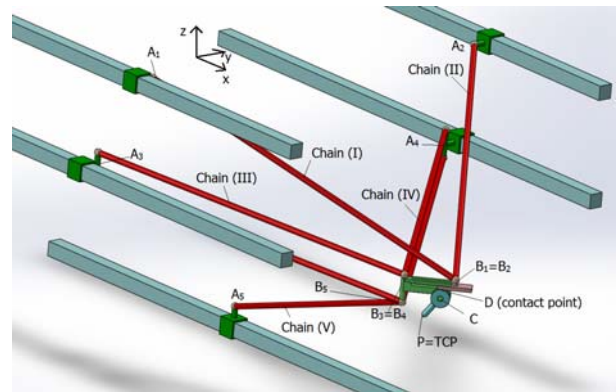
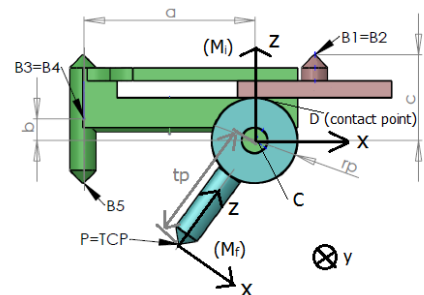


Figure 2. Simplified CAD for clarification purpose only. The base frame axes x,y, and z are shown on the drawing together with point notations.



(Mi): moving frame connected to 1st part of platform (green) with origin C  
(Mf): moving frame connected to the tool (blue) with origin P=TCP  
y-axis for (Mi) & (Mf) is perpendicular to the plane of the figure.

Figure 3. Platform dimensions and moving frames: ( $M_i$ ) and ( $M_f$ ).

- $L_i$  ( $i=1\dots5$ ): length of  $i$ -th arm with  $L_1 = L_2$ , and  $L_3 = L_4$ .

- $A_i = (x_i \ y_i \ z_i)^T = (q_i \ y_i \ z_i)^T$ ,  $i=1\dots5$ : with  $-y_1 = y_2 = L_{y1}$ ,  $-y_3 = y_4 = L_{y2}$ ,  $y_5 = 0$ ,  $z_1 = z_2 = L_{z1}$ ,  $z_3 = z_4 = 0$ , and  $z_5 = -L_{z2}$ .

- $B_i = (x_{bi} \ y_{bi} \ z_{bi})^T$  ( $i=1\dots5$ ): coordinates of  $B_i$  in the base frame. Note that  $N^m = (x_N^m \ y_N^m \ z_N^m)^T$  is the coordinates of an arbitrary point  $N$  with respect to moving frame  $m$  with  $m = M_i, M_f$ . Note that regarding the parallelogram arms, the points  $A_i$  and  $B_i$  are those along the imaginary mid-axes.

- Note that point  $B_5$  is along the vertical line passing through  $B_3 \equiv B_4$ , and it is not coincident with them in practice (it is practically not feasible as it causes collisions), however for simplifying the modelling, we assume that  $B_3 \equiv B_4 \equiv B_5$  and we compensate the real vertical offset  $\|B_4 B_5\|_{real}$  in the  $z$ -ordinate of point  $A_5$ . This also means that the triple spherical joint shown in Fig. 2 (which is intended as simple clarifying CAD drawing) can also be replaced (practically) by double spherical joint or its equivalent (connecting the lower rods of parallelograms (III) and (IV) to platform) and a third spherical joint or universal joint for connecting chain (V) to the platform: so the technical difficulty of realization of a triple spherical joint can be overcome.

- $P = (x \ y \ z)^T$ : is the TCP position. The robot's pose is  $x = (x \ y \ z \ \theta_z \ \theta_y)^T$  with  $\theta_z$  the rotation about the  $z$ -axis of the moving frame ( $M_i$ ) and  $\theta_y$  is the rotation angle about the  $y$ -axis of the moving frame ( $M_f$ ). Denote  $\theta = (\theta_z \ \theta_y)^T$ .

- Denote by  $q = (q_1 \ \dots \ q_5)^T$  the actuated joints' positions vector and by  $\dot{q} = (\dot{q}_1 \ \dots \ \dot{q}_5)^T$  the joints' velocities vector.

- $e_x$ ,  $e_y$ , and  $e_z$ : are the unit vectors along  $x$ ,  $y$  and  $z$  axes of the base frame, respectively.

- Denote by  $e_n = (\cos(\theta_z) \ \sin(\theta_z) \ 0)^T$  the unit vector along  $x$ -axis of frame ( $M_i$ ) and by  $e_f = (-\sin(\theta_z) \ \cos(\theta_z) \ 0)^T$  the unit vector along  $y$ -axis of ( $M_i$ ) and ( $M_f$ ).

- Denote  $v = \dot{P}$ ,  $\dot{\theta} = (\dot{\theta}_z \ \dot{\theta}_y)$ , and  $\dot{x} = (v^T \ \dot{\theta}^T)^T$ .

- Denote  $\Omega = [e_z \ e_f]$  and  $\Omega_1 = [e_z \ \mathbf{0}_{3 \times 1}]$ , then  $w = \Omega \dot{\theta}$  (angular velocity of the tool (blue color) expressed in the base frame) and  $w_1 = \Omega_1 \dot{\theta}$  (angular velocity of 1<sup>st</sup> part of platform (green color) and of the rack expressed in the base frame).

- $R_z$ , and  $R_y$  are the rotation matrices about  $z$ , and  $y$  axes respectively. We also define:  $R = R_z R_y$  the rotation matrix of the tool frame ( $M_f$ ) with respect to the base frame.

- Platform parameters ( $a$ ,  $b$ ,  $c$ ,  $r_p$  and  $t_p$ ) are shown on Fig. 3.

### III. THE INVERSE GEOMETRIC MODEL (IGM)

After having defined the mechanism's geometric elements, it is time to establish the IGM.

The robot's pose  $x$  is known, so we proceed by getting the points  $C$  and  $D$  (**contact point between rack and pinion**) as follows:

$$C = P + R C^{M_f} \quad (1)$$

and

$$D = C + r_p e_z \quad (2)$$

where  $C^{M_f}$ , and  $r_p$  are known (refer to Fig. 3). As we have now  $C$ , it is easy to get points  $B_3, \dots, B_5$  by:

$$B_3 \equiv B_4 \equiv B_5 = C + R_z B_3^{M_i}, \text{ where } B_3^{M_i} \equiv B_4^{M_i} \equiv B_5^{M_i} \quad (3)$$

It is important to recall here that point  $B_5$  is practically on the vertical line passing through  $B_3 \equiv B_4$  but not coincident with them (to avoid collisions in practical case) as shown clearly in Fig. 2-3, but for simplifying the modelling we considered them coincident by compensating the real vertical offset in the  $z$ -ordinate of point  $A_5$  (as we already explained in the previous section).

Now to get the point  $B_1 \equiv B_2$ , we use the following relation:

$$B_1 \equiv B_2 = D + r_p \theta_y e_h + (c - r_p) e_z \quad (4)$$

As we have obtained all the points  $B_i$ , we use the following relation to get  $A_i$ :

$$A_i B_i^2 = L_i^2, \forall i = 1\dots5 \quad (5)$$

Relation (5) gives:

$$q_i = x_i = x_{bi} \pm \sqrt{L_i^2 - (y_{bi} - y_i)^2 - (z_{bi} - z_i)^2}, \forall i = 1\dots5 \quad (6)$$

We choose the assembly mode to satisfy:

$$q_i = x_i \leq x_{bi}, \forall i = 1\dots5 \quad (7)$$

The solution is then:

$$q_i = x_i = x_{bi} - \sqrt{L_i^2 - (y_{bi} - y_i)^2 - (z_{bi} - z_i)^2}, \forall i = 1\dots5 \quad (8)$$

Hence, the IGM of the new mechanism has been established.

#### IV. THE DIRECT GEOMETRIC MODEL (DGM)

The direct geometric model DGM is most often difficult to establish analytically for parallel robots. However, in our case it is quite simple. In fact, we have all the coordinates of points  $A_i, \forall i=1\dots 5$ , so we get first the coordinates  $B_3 \equiv B_4 \equiv B_5$ , then  $B_1 \equiv B_2$ . Due to space limitation, we will describe it briefly. Consider the following system of equations:

$$\begin{cases} A_3 B_3^2 = L_3^2 \\ A_4 B_4^2 = L_4^2, \text{ where } B_3 \equiv B_4 \equiv B_5 \\ A_5 B_5^2 = L_5^2 \end{cases} \quad (9)$$

Solving the intersection of the three spheres described by (9), we get in general two possible solutions, call them  $B_3^{s1}$  and  $B_3^{s2}$ . Then, the z-ordinate of the point  $B_1 \equiv B_2$  is known and it is given by (refer to Fig.3):

$$z_{b1} \equiv z_{b2} = z_{b3} + (c - b) \quad (10)$$

Hence, we have two possible z-ordinates  $z_{b1}^{s1}$  and  $z_{b1}^{s2}$  corresponding to  $B_3^{s1}$  and  $B_3^{s2}$  respectively. We also have:

$$\begin{cases} A_1 B_1^2 = L_1^2 \\ A_2 B_2^2 = L_2^2 \end{cases}, \text{ where } B_1 \equiv B_2 \quad (11)$$

Again, substituting (10) in (11), we get in general two solutions corresponding to each value of  $z_{b1}$ , we call them  $B_1^{s11}, B_1^{s12}$  (corresponding to  $z_{b1}^{s1}$ ) and  $B_1^{s21}, B_1^{s22}$  (corresponding to  $z_{b1}^{s2}$ ). Thus, we have in general a set of four solutions, denote it by  $\mathcal{S} = \{S_1, S_2, S_3, S_4\}$  with  $S_1 = \langle B_3^{s1}, B_1^{s11} \rangle$ ,  $S_2 = \langle B_3^{s1}, B_1^{s12} \rangle$ ,  $S_3 = \langle B_3^{s2}, B_1^{s21} \rangle$ , and  $S_4 = \langle B_3^{s2}, B_1^{s22} \rangle$ . Among them, a unique solution which satisfies the assembly mode (condition (7)) exists and as a result we have all the points  $B_i, \forall i=1\dots 5$ . Consider then the vector  $\eta$  which is the projection of  $B_3 B_1$  in the xy-plane, i.e.:

$$\eta = (\eta_x \quad \eta_y \quad 0)^T = B_3 B_1 - (B_3 B_1^T e_z) e_z \quad (12)$$

Then, we have  $R_z$  knowing:

$$\theta_z = \text{atan} 2(\eta_y, \eta_x) \in [-\pi; \pi] \quad (13)$$

We then get:

$$\begin{cases} C = B_3 + R_z B_3 C^{M_i} \\ D = C + r_p e_z \end{cases} \quad (14)$$

The matrix  $R = R_z R_y$  is then computed where  $R_y$  is known after getting  $\theta_y$  via the following relation:

$$\theta_y = \frac{(DB_1 - (c - r_p) e_z)^T e_h}{r_p} \quad (15)$$

The TCP position is given by:

$$P = (x \quad y \quad z)^T = C - R C^{M_f} \quad (16)$$

The pose is then given by  $x = (x \quad y \quad z \quad \theta_z \quad \theta_y)^T$  and the DGM is therefore established.

#### V. THE JACOBIANS

In order to study singularities and evaluate the performance of the robot, we need to establish the Jacobian matrix  $J$  or the inverse Jacobian matrix  $J_m$ . Thus, we need to get the relation between  $\dot{q}$  and  $\dot{x} = (\dot{x} \quad \dot{y} \quad \dot{z} \quad \dot{\theta}_z \quad \dot{\theta}_y)^T$  where  $\dot{\theta}_z$  and  $\dot{\theta}_y$  are the angular velocities about the two perpendicular directions  $e_z$  and  $e_f$  (note that in this case geometric inverse and forward Jacobians are essentially the same as their corresponding analytical notions; for discussion on this matter refer to [11]).

So, we proceed by differentiating (5) with respect to time to get:

$$A_i B_i^T v_{A_i} = A_i B_i^T v_{B_i}, \quad \forall i=1\dots 5 \quad (17)$$

We have:

$$v_{A_i} = \dot{q}_i e_x, \quad \forall i=1\dots 5 \quad (18)$$

It remains to get  $v_{B_i}$  in terms of  $\dot{x}$ . For  $i=1, 2$ , we have:

$$\begin{cases} v_{B_i} = v_D + w_1 \times DB_i = v + w \times PD + w_1 \times DB_i \\ = v - \widehat{PD} \Omega \dot{\theta} - \widehat{DB_i} \Omega_i \dot{\theta} \\ = \left[ I_{3 \times 3} \quad -(\widehat{PD} \Omega \dot{\theta} + \widehat{DB_i} \Omega_i) \right] \dot{x}, \quad \forall i=1, 2 \\ I_{3 \times 3}: 3 \times 3 \text{ identity matrix} \\ \widehat{N} = N \times \dots: \text{pre-cross product matrix} \end{cases} \quad (19)$$

Regarding  $v_{B_3} \equiv v_{B_4} \equiv v_{B_5}$ , we have for  $i=3, 4, 5$ :

$$\begin{aligned} v_{B_i} &= v_C + w_1 \times CB_i = v + w \times PC + w_1 \times CB_i \\ &= \left[ I_{3 \times 3} \quad -(\widehat{PC} \Omega + \widehat{CB_i} \Omega_i) \right] \dot{x} \end{aligned} \quad (20)$$

Regrouping (17) in a matrix form and using (18) through (20), we get:

$$J_q \dot{q} = J_x \dot{x} \quad (21)$$

The matrix  $J_q$  is given by:

$$\begin{cases} J_q = \text{diag}(n_1^T e_x, \dots, n_5^T e_x), \dim(J_q) = 5 \times 5, \\ n_i = A_i B_i, \quad \forall i=1\dots 5 \end{cases} \quad (22)$$

As for  $J_x$ , it is as follows:

$$\left\{ \begin{array}{l} \mathbf{J}_x = \begin{bmatrix} \mathbf{n}_1^T & -\mathbf{n}_1^T (\widehat{PD} \boldsymbol{\Omega} + \widehat{DB}_1 \boldsymbol{\Omega}_1) \\ \mathbf{n}_2^T & -\mathbf{n}_2^T (\widehat{PD} \boldsymbol{\Omega} + \widehat{DB}_2 \boldsymbol{\Omega}_1) \\ \mathbf{n}_3^T & -\mathbf{n}_3^T (\widehat{PC} \boldsymbol{\Omega} + \widehat{CB}_3 \boldsymbol{\Omega}_1) \\ \mathbf{n}_4^T & -\mathbf{n}_4^T (\widehat{PC} \boldsymbol{\Omega} + \widehat{CB}_4 \boldsymbol{\Omega}_1) \\ \mathbf{n}_5^T & -\mathbf{n}_5^T (\widehat{PC} \boldsymbol{\Omega} + \widehat{CB}_5 \boldsymbol{\Omega}_1) \end{bmatrix} \\ \dim(\mathbf{J}_x) = 5 \times 5 \end{array} \right. \quad (23)$$

When  $\mathbf{J}_q^{-1}$  exists ( $\det(\mathbf{J}_q) \neq 0$ ), the inverse Jacobian  $\mathbf{J}_m$  ( $\dot{\mathbf{q}} = \mathbf{J}_m \dot{\mathbf{x}}$ ) is given by:

$$\mathbf{J}_m = \mathbf{J}_q^{-1} \mathbf{J}_x, \dim(\mathbf{J}_m) = 5 \times 5 \quad (24)$$

Similarly, when  $\mathbf{J}_x^{-1}$  exists, the forward Jacobian  $\mathbf{J}$  ( $\dot{\mathbf{x}} = \mathbf{J} \dot{\mathbf{q}}$ ) is given by:

$$\mathbf{J} = \mathbf{J}_m^{-1} \mathbf{J}_x^{-1} \mathbf{J}_q, \dim(\mathbf{J}) = 5 \times 5 \quad (25)$$

So far, we have established the forward and inverse Jacobian matrices  $\mathbf{J}$  and  $\mathbf{J}_m$ , after establishing the matrices  $\mathbf{J}_q$  and  $\mathbf{J}_x$ . In what follows, we discuss the series and parallel type singularities of the mechanism at hand.

#### A. Series Type Singularities

Series-type singularity occurs when  $\mathbf{J}_q$  becomes rank deficient i.e. when  $\det(\mathbf{J}_q) = 0$ ; in such a case the end-effector might be fixed ( $\dot{\mathbf{x}} = \mathbf{0}$ ), but the actuators are capable of infinitesimal motion ( $\dot{\mathbf{q}} \neq \mathbf{0}$ ). This means:

$$\det(\mathbf{J}_q) = 0 \Leftrightarrow \exists i_o \in \{1, \dots, 5\}; \mathbf{n}_{i_o}^T \mathbf{e}_x = 0 \Leftrightarrow \mathbf{n}_{i_o} \perp \mathbf{e}_x \quad (26)$$

Relation (26) shows that series singularity occurs when there is a completely stretched arm (arm in the yz plane) which implies that singularity if exists (corresponding pose is accessible), it will be on the boundary of the geometrically accessible region and thus forms no problem.

#### B. Parallel Type Singularities

As for parallel-type singularities, these occur when the matrix  $\mathbf{J}_x$  becomes rank deficient; in such a case the actuators can be fixed ( $\dot{\mathbf{q}} = \mathbf{0}$ ), but still the end-effector is capable of infinitesimal motion ( $\dot{\mathbf{x}} \neq \mathbf{0}$ ). We will do linear operations on  $\mathbf{J}_x$  to facilitate the study. The first operation is a change of the TCP to be confounded with  $C$  (this does not change the rank since it is mathematically described as adding linear combinations of some columns to a certain column)<sup>(1)</sup>, and thus we obtain the following matrix:

<sup>(1)</sup> Adding  $(\mathbf{n}_1^T (\mathbf{PC} \times \mathbf{e}_z) \dots \mathbf{n}_5^T (\mathbf{PC} \times \mathbf{e}_z))^T$  (linear combinations of first three columns) to 4<sup>th</sup> column of  $\mathbf{J}_x$  and adding  $(\mathbf{n}_1^T (\mathbf{PC} \times \mathbf{e}_f) \dots \mathbf{n}_5^T (\mathbf{PC} \times \mathbf{e}_f))^T$  to 5<sup>th</sup> column of  $\mathbf{J}_x$ .

$$\left\{ \begin{array}{l} \mathbf{M} = \begin{bmatrix} \mathbf{n}_1^T & -\mathbf{n}_1^T (\widehat{CD} \boldsymbol{\Omega} + \widehat{DB}_1 \boldsymbol{\Omega}_1) \\ \mathbf{n}_2^T & -\mathbf{n}_2^T (\widehat{CD} \boldsymbol{\Omega} + \widehat{DB}_2 \boldsymbol{\Omega}_1) \\ \mathbf{n}_3^T & -\mathbf{n}_3^T (\widehat{CB}_3 \boldsymbol{\Omega}_1) \\ \mathbf{n}_4^T & -\mathbf{n}_4^T (\widehat{CB}_4 \boldsymbol{\Omega}_1) \\ \mathbf{n}_5^T & -\mathbf{n}_5^T (\widehat{CB}_5 \boldsymbol{\Omega}_1) \end{bmatrix} \\ = \begin{bmatrix} \mathbf{n}_1^T & r_p \theta_y \mathbf{n}_1^T \mathbf{e}_f & r_p \mathbf{n}_1^T \mathbf{e}_h \\ \mathbf{n}_2^T & r_p \theta_y \mathbf{n}_2^T \mathbf{e}_f & r_p \mathbf{n}_2^T \mathbf{e}_h \\ \mathbf{n}_3^T & -a \mathbf{n}_3^T \mathbf{e}_f & 0 \\ \mathbf{n}_4^T & -a \mathbf{n}_4^T \mathbf{e}_f & 0 \\ \mathbf{n}_5^T & -a \mathbf{n}_5^T \mathbf{e}_f & 0 \end{bmatrix} \end{array} \right. \quad (27)$$

Regarding the parameters  $a$  and  $r_p$ , refer to Fig. 3. To simplify the rank study of  $\mathbf{M}$ , we make another linear operation by adding  $(a \mathbf{n}_1^T \mathbf{e}_f \dots a \mathbf{n}_5^T \mathbf{e}_f)^T$  to 4<sup>th</sup> column of  $\mathbf{M}$  which gives us the simplified matrix:

$$\mathbf{N} = \begin{bmatrix} \mathbf{n}_1^T & (a + r_p \theta_y) \mathbf{n}_1^T \mathbf{e}_f & r_p \mathbf{n}_1^T \mathbf{e}_h \\ \mathbf{n}_2^T & (a + r_p \theta_y) \mathbf{n}_2^T \mathbf{e}_f & r_p \mathbf{n}_2^T \mathbf{e}_h \\ \mathbf{n}_3^T & 0 & 0 \\ \mathbf{n}_4^T & 0 & 0 \\ \mathbf{n}_5^T & 0 & 0 \end{bmatrix} \quad (28)$$

Then, it is sufficient to study the rank of the following two sub-matrices of  $\mathbf{N}$ :

$$\left\{ \begin{array}{l} \mathbf{N}_1 = [\mathbf{n}_3 \quad \mathbf{n}_4 \quad \mathbf{n}_5]^T \\ \mathbf{N}_2 = \begin{bmatrix} (a + r_p \theta_y) \mathbf{n}_1^T \mathbf{e}_f & r_p \mathbf{n}_1^T \mathbf{e}_h \\ (a + r_p \theta_y) \mathbf{n}_2^T \mathbf{e}_f & r_p \mathbf{n}_2^T \mathbf{e}_h \end{bmatrix} \end{array} \right. \quad (29)$$

The matrix  $\mathbf{N}$  (equivalently  $\mathbf{M}$  and  $\mathbf{J}_x$ ) is singular when any of the two matrices  $\mathbf{N}_1$  or  $\mathbf{N}_2$  is singular (rank deficient). The matrix  $\mathbf{N}_1$  is of full rank as long as the three arms, namely (III), (IV) and (V), are not coplanar, which is guaranteed within the geometrically accessible space excluding its boundary. Actually, the matrix  $\mathbf{N}_1$  (equivalently  $\mathbf{J}_x$ ) may become rank deficient only in one case; this is when all the aforementioned arms are in one plane which cannot be -due to the assembly mode (condition (7))- except the yz-plane and hence parallel-type singularity confounded with serial-type singularity which cannot occur except at boundary of the geometrically accessible workspace-provided that the corresponding pose is geometrically accessible. Regarding the rank of  $\mathbf{N}_2$ , computing and simplifying its determinant, we get:

$$\begin{aligned} \det(\mathbf{N}_2) &= (a + r_p \theta_y) (n_{1x} n_{2y} - n_{1y} n_{2x}) (e_{fx} e_{hy} - e_{fy} e_{hx}) \\ &= -(a + r_p \theta_y) (n_{1x} n_{2y} - n_{1y} n_{2x}) \end{aligned} \quad (30)$$

Note that  $vect_x$  and  $vect_y$  mean x and y ordinates of vector  $vect$  respectively. We have  $a + r_p \theta_y \neq 0$ ,  $\forall \theta_y \in [-45^\circ; +45^\circ]$  (i.e.  $a > (r_p \pi)/4$ ). Then, the only case where  $N_2$  (equivalently  $J_x$ ) is singular is when  $(n_{1x} n_{2y} - n_{1y} n_{2x}) = 0$ , meaning that when the projections of arms (I) and (II) in the xy plane are collinear which is not possible except when both arms are in the yz-plane and thus a parallel-type singularity coincident with a series-type singularity. Again, such a situation cannot occur except on the boundary of the geometrically accessible workspace-if the corresponding pose is geometrically accessible in the first place.

So, in brief, in this section we have proved that both types of singularities if to exist they are necessarily on the boundary of the geometrically accessible workspace-provided that the corresponding poses are geometrically reachable-and thus form no problem. Also, any parallel-type singularity if present, it exists confounded with a series-type singularity. As a result, we guarantee the absence of singularities of all types within the geometrically accessible workspace excluding its boundary.

## VI. WORKSPACE ANALYSIS

In this section, we present sample workspace analysis based on isotropic force-velocity performance index  $FVI$  introduced in [3] that the reader may refer to. The index is defined as follows:

$$FVI = \min \left( \frac{v_w}{v_{wl}}, \frac{f_w}{f_{wl}} \right) \quad (31)$$

The terms  $v_w$  and  $f_w$  are the worst speed and the worst force <sup>(2)</sup> respectively, whereas  $v_{wl}$  and  $f_{wl}$  are the desired lower bounds for the worst speed and worst force respectively. Actually,  $v_w$  is nothing except the largest isotropic speed (radius of the largest sphere included in the zonotope of the operational velocities), and  $f_w$  is similarly the largest isotropic force (radius of largest sphere included in the operational force zonotope (refer to footnote <sup>(2)</sup>)). In case of non-redundant robot, the pseudo-inverse is replaced by the usual matrix inverse. Actually, this index has been developed to overcome the loss of significance of singular values and based-upon indices (such as condition number, manipulability index, etc.) in case of redundant robots, but it is applicable to all types of robots and holds a concrete physical significance quantifying the maximal attainable

<sup>(2)</sup>  $f_w$  is calculated considering minimum norm torque vector solution of  $f = J_m^T \tau$  i.e. considering the joint torques vector  $\tau$  satisfying  $[\text{null}(J_m^T)]^T \tau = \mathbf{0}$  and thus having:  $\tau = J^T f = J_m^{*T} f$  with  $J = J_m^*$  the pseudo-inverse of  $J_m$  (case of redundancy) and  $f$  the operational force vector. Note to have physical significance and consistency of  $f_w$ , the matrix  $J_m$  must be homogeneous.

speed and maximal static force that can be supported by the end-effector regardless of direction. Furthermore, it serves as well as a singularity measure (closeness to singularity when  $FVI \rightarrow 0$ )<sup>(3)</sup>, although this latter significance is not our major concern or focus here as detailed analytical singularity analysis has been done in the previous section.

Since we have mixed dofs (translation and rotation), it is mandatory to homogenize  $J_m$  using a suitable technique before evaluating the index at each pose. Then, we consider the homogeneous inverse Jacobian matrix  $J_{mw} = J_m W^{-1}$  and its pseudo-inverse  $J_w$  (in case of redundancy) or its inverse (i.e. the homogenized forward Jacobian  $J_w = W J = W J_m^{-1}$ ) as in our case here (non-redundancy). Matrix  $W$  is the weighing or homogenization matrix and it is chosen here to be  $W = \text{diag}(1, 1, 1, a, t_p)$  (the choice of the homogenization matrix is arbitrary with the condition that the weighing/homogenization matrix should assure that the initial (non-homogenized) and the corresponding homogenized matrices are of same rank where  $W$  is a square matrix: in our case we use the characteristic lengths technique). We then have:

$$\begin{cases} v_w = \min_{i=1..m} \left( \frac{1}{\|j_{mw_i}\|} \right) \dot{q}_{\max} \\ f_w = \min_{i=1..m} \left( \frac{1}{\|j_{wc_i}\|} \right) \tau_{\max} \\ m : \text{number of actuators} \end{cases} \quad (32)$$

where  $j_{mw_i}$  and  $j_{wc_i}$  are the i-th row vector and i-th column vector of  $J_{mw}$  and  $J_w$  respectively.

In the figures that follow, we present the sample workspace plots and evaluation for the following geometric parameters (non-optimized parameters):  $L_i = 1m$ ,  $\forall i = 1..5$ ,  $L_{y1} = L_{y2} = 0.5m$ ,  $L_{z1} = 0.3m$ ,  $L_{z2} = 0.5m$ ,  $a = 0.1m$ ,  $b = 0.0875m$ ,  $c = 0.125m$ ,  $r_p = 0.025m$ ,  $t_p = 0.075m$ ,  $t_c = 0.0175m$ , and  $t_l = 0.2m$ . Note that  $t_c$  and  $t_l$  are the minimum offsets from the sliders' planes in case of zero rotation and full range of rotations, respectively.

Regarding the workspace plots, we have evaluated the yz-regions for zero orientation ( $\theta_z = \theta_y = 0^\circ$ ) on one hand and for full range rotations (i.e.  $\theta_z \in [-90^\circ; +90^\circ]$  and  $\theta_y \in [-45^\circ; +45^\circ]$ ) on the other hand; this is since the inverse Jacobian is independent of x position. In case of full range orientation, we evaluated the index  $FVI$  for  $\theta_z = \pm(90^\circ, 60^\circ, 45^\circ, 30^\circ, 0^\circ)$  and  $\theta_y = \pm(45^\circ, 30^\circ, 0^\circ)$  being frequently used orientations for the sake of reducing computation time, and the worst value of  $FVI$  (the lowest

<sup>(3)</sup> Note that when  $v_w \rightarrow 0$ , it means approaching a series-type singularity, while having  $f_w \rightarrow 0$ , it means approaching a parallel-type singularity.



value) has been associated with the corresponding  $(y, z)$  position. We have done the study on homogenized inverse Jacobian matrix  $\mathbf{J}_{mw}$  (see Fig. 4-5) and a study considering only translational motion by considering the translational parts <sup>(4)</sup> of  $\mathbf{J}_m$  and  $\mathbf{J}$ , called  $\mathbf{J}_{mp}$  and  $\mathbf{J}_p$  respectively, which is the most interesting for us (regarding translational speed capability and pure static force capacity: see Fig. 6-7) as former study on  $\mathbf{J}_{mw}$  can change with the chosen homogenization matrix  $\mathbf{W}$  and its only importance is mainly as singularity performance measure (there is no real physical significance of studying the norm of 5 dimensional velocity vector composed by linear translation of a point and tangential velocity of another point with respect to the TCP; this is always the case when having mixed dofs irrespective of the index used). Note that considering translational performances only implies redundancy (3 dofs (3T) and 5 actuators, i.e.  $\mathbf{J}_{mp}$  and  $\mathbf{J}_p$  are not square matrices). Note that the case of studying translational motion only implies that the angular velocity is controlled to be zero and there are no torques applied on the platform-i.e. we only have pure forces at the TCP). Also, it is worth mentioning that if singular values are significant regarding  $\mathbf{J}_{mw}$  being a square matrix, singular values of  $\mathbf{J}_{mp}$  and  $\mathbf{J}_p$  are no more significant regarding output performance capabilities as explained in [12].

In all the plots, we have set  $v_{wl} = \dot{q}_{\max}$  and  $f_{wl} = \tau_{\max}$  as to compare the isotropic velocity and force output performances of the mechanism relative to the maximal performances in velocity and force of the individual actuator (all actuators are considered identical). Note also that in the figures, the red solid lines show the collision limits with the sliders' planes, while the dashed black box is the projection of the robot's frame on the  $yz$  plane. The red lines are offset from vertical black dashed lines by  $t_c$  in case of zero rotation and by  $t_l$  in case of full rotation.

Finally, to conclude up the section, it is clear that the mechanism has large singularity-free workspace considering zero rotation and full rotational capabilities. As a matter of fact, if we consider the ratio of the  $yz$  accessible region (with or without rotation) to the area of the black dashed box representing the frame of the robot, this ratio is clearly high.

Besides, the kinetostatic performances are quite interesting (remember that these represent capabilities that can be achieved in all directions). It would even be more improved and its performance more enhanced after properly optimizing this mechanism which is not our concern here. Additionally, the workspace is convex; such convexity is important regarding trajectory planning where each two points can be joined by a straight line trajectory.

<sup>(4)</sup> In this case  $\mathbf{J}_{mw}$  and  $\mathbf{J}_w$  in relation (32) are respectively replaced by  $\mathbf{J}_{mp} = \mathbf{J}_m(:, 1:3)$  i.e. first three columns of  $\mathbf{J}_m$  and  $\mathbf{J}_p = \mathbf{J}(1:3, :)$  i.e. first three rows of  $\mathbf{J} = \mathbf{J}_m^{-1}$  (non-redundant robot) and  $\mathbf{J} = \mathbf{J}_m^*$  (pseudo-inverse of  $\mathbf{J}_m$  in case of redundancy).

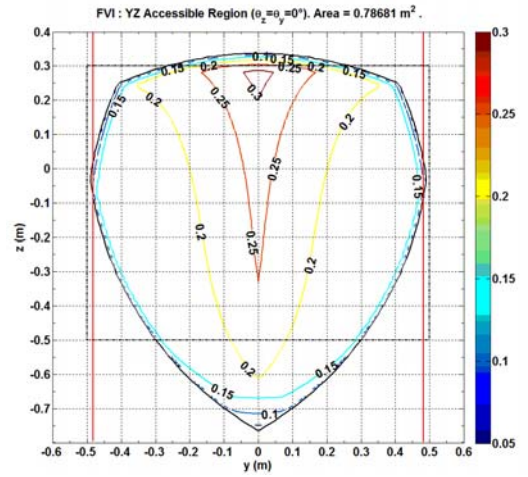


Figure 4. Geometrically accessible workspace for zero rotation: evaluated using index  $FVI(\mathbf{J}_{mw})$ .

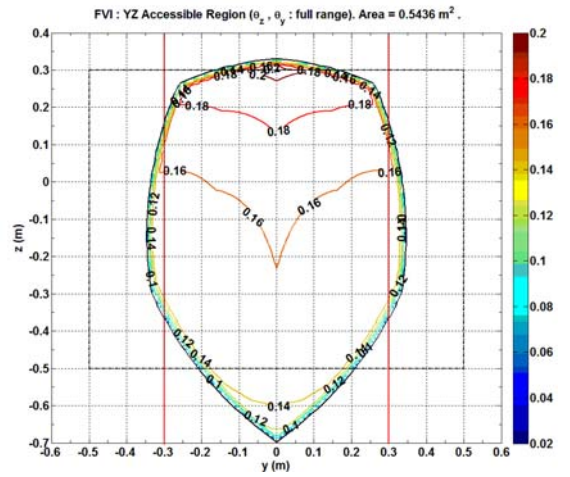


Figure 5. Geometrically accessible workspace with full rotational range: evaluated using  $FVI(\mathbf{J}_{mw})$ .

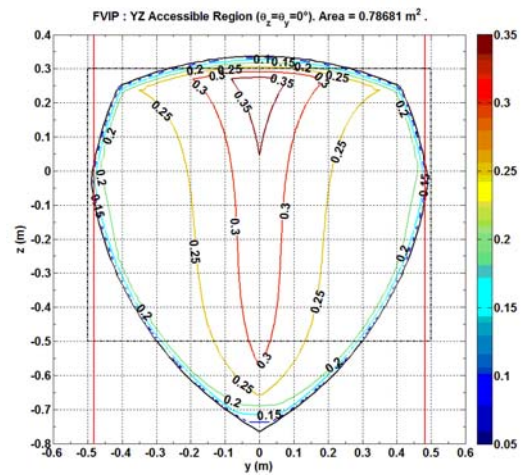


Figure 6. Geometrically accessible workspace for zero rotation: evaluated using index FVIP (the P indicates study of positional parts of inverse Jacobian  $\mathbf{J}_m$  and forward Jacobian  $\mathbf{J}$  matrices).



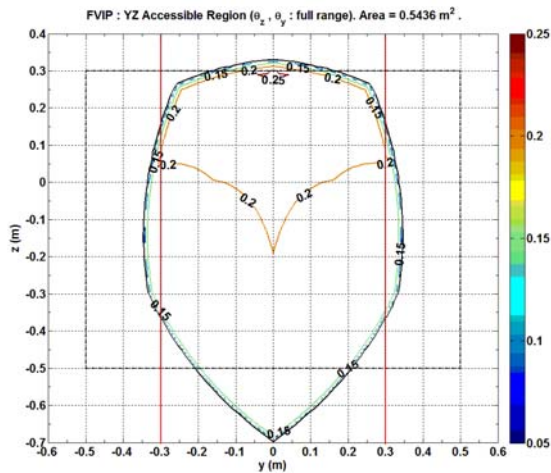


Figure 7. Geometrically accessible workspace with full rotational range: evaluated using index FVIP (the P indicates study of positional parts of inverse Jacobian  $J_m$  and forward Jacobian  $J$  matrices).

## VII. CONCLUSION AND PERSPECTIVES

In this paper, we have introduced a novel 5 dofs (3T-2R) parallel mechanism that has the following characteristics:

- The functioning of the mechanism is simple.
- It admits an analytical direct geometric model DGM that is a rare feature to find regarding parallel manipulators and can be very helpful regarding control issues.
- It has independent motion along x-direction only restricted by available stroke for prismatic joints.
- Parallel type singularities if to exist, they are coincident with series-type singularities and cannot occur except at the geometrically accessible workspace boundary (provided that corresponding poses are geometrically achievable). Thus, in any case they do not form any problem.
- It has large singularity-free cross-sectional workspace with or without rotation. Also, its isotropic kinetostatic performance is quite interesting and it is expected to be improved even more after optimization. This has been clarified by the preliminary analysis of workspace for non-optimized set of geometric parameters.
- High stiffness having all the limbs sharing in the load support along all directions.
- Having spherical joints at the arms' extremities puts these arms under tension/compression forces which improves accuracy due to reduced deformation in the arms.

It is important to mention that this mechanism is a starting point, as other versions based on this one and with better performance are being thought of and may be the subject for future publications. We also emphasize that in this mechanism, the use of rack-pinion is meant to simplify the modelling and the preliminary presentation of the mechanism, but in the realistic case it would be replaced by

more rigid linear-to-rotational motion transformation mechanism.

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